

# Surrogate modelling for high-lift multi-element hydrofoil shape optimization of a hydrokinetic turbine blade

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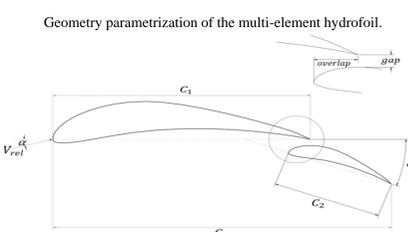
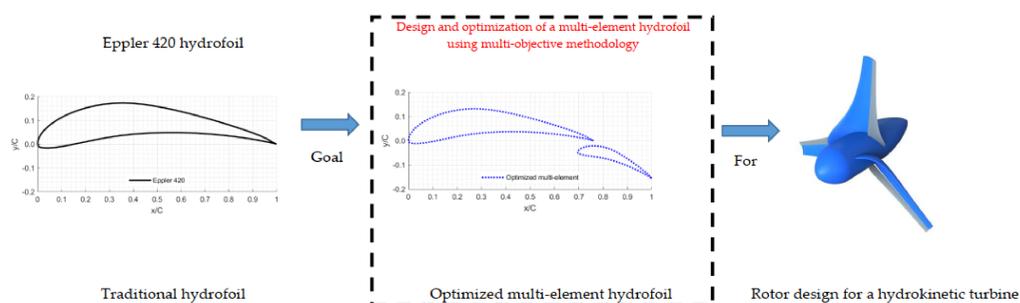
## Introduction

The hydrodynamic shape of a blade is one of the most important factors in the design process of a horizontal-axis hydrokinetic turbine influencing its performance. The present work is focused on the design and the hydrodynamic analysis of a high-lift system using the optimization method of surrogate models and computational fluid dynamics (CFD).

The parameters affecting the amount of the lift and the drag force that a hydrofoil can generate are the gap, the overlap, the flap deflection angle ( $\delta$ ), the flap chord length ( $C_2$ ) and the angle of attack ( $\alpha$ ) of the hydrofoil. These factors were varied to examine the turbine performance in terms of the ratio between the lift ( $C_L$ ) and the drag coefficient ( $C_D$ ), and the minimum negative pressure coefficient ( $\min C_{pre}$ ) in order to avoid the cavitation inception. For this purpose, surrogate models were implemented to analyse the CFD results and find the optimal combination of the design parameters involved in the high-lift hydrofoil. The traditional Eppler 420 hydrofoil was utilized for the design of the multi-element profile, which was composed of a main element and a flap. The multi-element design selected as optimal had a gap of 2.825 % $C_1$ , an overlap of 8.52 % $C_1$ , a  $\delta$  of 19.765°, a  $C_2$  of 42.471 % $C_1$  and a  $\alpha$  of -4°, where  $C_1$  refers to the chord length of the main element. In comparison with the traditional Eppler 420 hydrofoil,  $C_L/C_D$  ratio increases from 39.050 to 42.517.

**Research problem:** Optimization of a two-dimensional multi-element hydrofoil shape for small horizontal-axis hydrokinetic turbines

## Material and methods



Parameter (units)	Values
C (m)	0.1773
$V_{rel} = U_{\infty}$ (m/s)	5.517
$\alpha$ (°)	$-10^\circ \leq \alpha \leq 10^\circ$
$\delta$ (°)	$10^\circ \leq \delta \leq 30^\circ$
gap (°)	$1\%C_1 \leq gap \leq 5\%C_1$
ovl (m)	$-5\%C_1 \leq ovl \leq 20\%C_1$
$C_2$ (m)	$30\%C_1 \leq C_2 \leq 75\%C_1$

### Objective

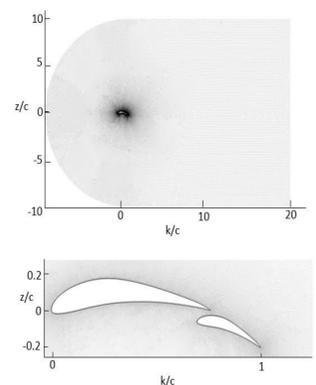
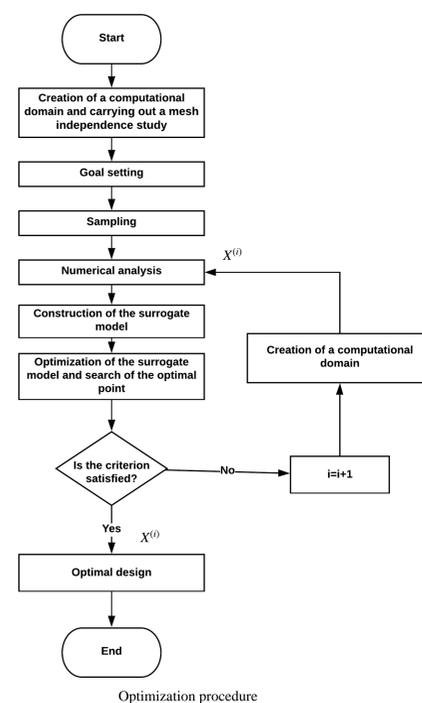
In this study, the objective functions were related to the maximization of the  $C_L$  and the minimization of  $C_D$  of the multi-element hydrofoil

$$\max C_L \text{ or } -\min C_L$$

$$\min C_D$$

$$|\min C_{pre}| \leq 4$$

where  $\min C_{pre}$  refers to the minimum negative pressure coefficient to avoid the cavitation inception.



A computational grid with a C-topology for an Eppler 420 multi-element hydrofoil. b) A view of the computational grid close to the Eppler 420 multi-element hydrofoil.

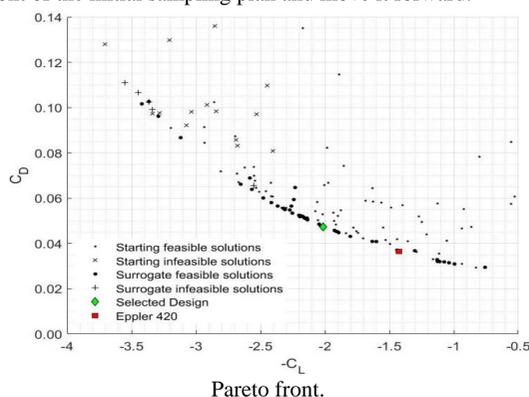
### Geometric hydrofoil and boundary condition specifications

Parameter (units)	Description
Blade profile	Eppler 420
Blade chord length (C)	0.177 m
Fluid	Water at 25 °C
Turbulence model	$k - \omega$ SST
Inlet	Velocity inlet
Outlet	Pressure outlet
Upper boundaries (top edge)	Symmetric boundary
Lower boundaries (bottom edge)	Symmetric boundary
Hydrofoil	No-slip wall

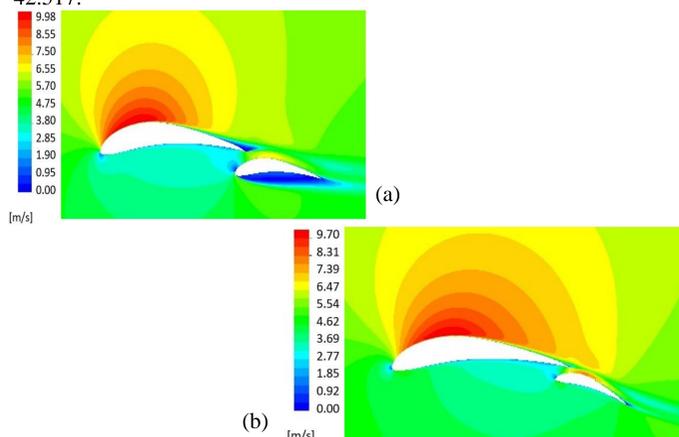
## Results and discussion

Through the established Surrogate Model (SM), a Pareto front was constructed, as shown in the figure below. In the figure, the results concerning the initial sampling, the design suggested by the SM, the starting design (Eppler 420 hydrofoil) and the selected multi-element design based on the  $C_L/C_D$  ratio are illustrated.

It is shown that few of the initial designs contributed to the Pareto front and some of them granted a better  $C_L/C_D$  ratio than that corresponding to the starting Eppler 420 hydrofoil. Additionally, the designs supplied by the SM contributed to the Pareto front with new designs that fill the gaps in the Pareto front of the initial sampling plan and move it forward.

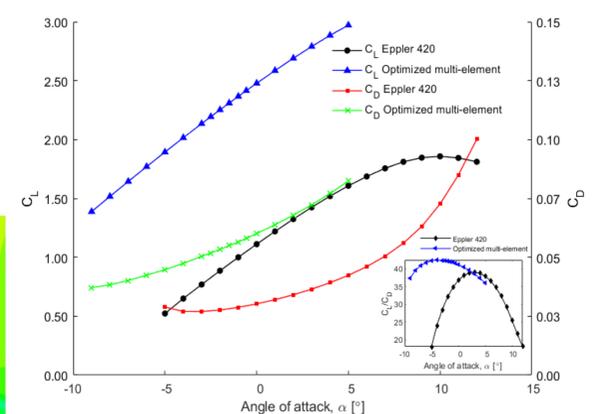


The initial and optimized multi-element hydrofoil shapes are represented in the figure below. It can be observed that the resulting flow was more aligned with the flap compared to the traditional hydrofoil. The multi-element design selected as the optimal one had a gap of 2.825 % $C_1$ , an overlap of 8.52 % $C_1$ , a  $\delta$  of 19.765°, a  $C_2$  of 42.471 % $C_1$  and a  $\alpha$  of -4°.  $C_L$  and  $C_D$  were equal to 2.016 and 0.047, respectively, providing a  $C_L/C_D$  of 42.517.



Contours of velocity of the flow around the initial (a) and optimized (b) multi-element hydrofoil.

The variation of  $C_L$ ,  $C_D$  and  $C_L/C_D$  with  $\alpha$  are shown in the figure below. It can be clearly found that there was a large performance improvement of the optimized shape in comparison with the initial one (i.e.,  $C_L/C_D$  increased from 39.050 to 42.517).



$C_L/C_D$  ratio as a function of  $\alpha$  for a traditional hydrofoil and for an optimized multi-element hydrofoil using the Eppler 420 profile.

## Conclusion

The design of a multi-element hydrofoil based on SM was presented in this study. The use of SM refers to an approach that can be employed for the design of hydrokinetic turbines, allowing the correction of a multi-element hydrofoil shape, aiming at preventing cavitation. In this study, the objective was to maximize  $C_L$  and minimize  $C_D$  subjected to  $C_{pre} < 4$  constraint. The results showed that the improvement of  $C_L/C_D$  was significant compared to the conventional hydrofoil. The multi-element hydrofoil had a  $C_L/C_D$  of 8.87% larger than that of the traditional hydrofoil.

The design of the optimal hydrofoil for hydrokinetic appliances always requires an amount of time experiments and computational analysis in order to achieve the planned goals. In this work, the SM allowed reducing the time of multi-element hydrofoil design process involved in the blade manufacture of a horizontal-axis hydrokinetic turbine due to the reduction of the iterations number and the CFD analysis within the optimization procedure.

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